



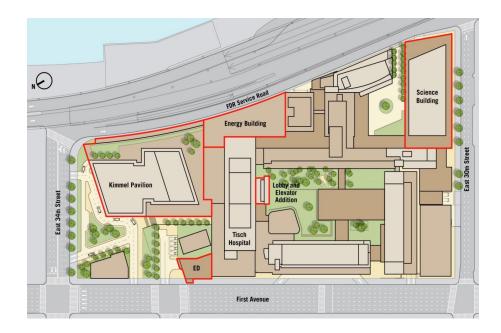
Control Strategies for Operating Rooms and Critical Spaces

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Introduction to NYULH

- 3.6 M SQFT Main Campus
- Centralized Steam and CHW systems
- 2 CHPs Primary 7 MW Gas Turbine w/ 2.4 MW Steam Turbine combined cycle
- Secondary 2.9 MW reciprocating gas engine CHP
- 3 Electric + 1 Steam Turbine Chiller Plant feeding single loop
- 4 Lab Buildings, 2 Vivaria, 67 ORs





Outline

- Overview of Critical Spaces and Importance of Critical Spaces to Energy Conservation
- Pressurization and Airflow
- Temperature and Humidity Control
- Controls Strategy



Critical Room Ventilation Requirements

• Air Change Rates (ACH)

- Specified minimum total and outdoor air change rates per hour
- Operating Rooms: 20 ACH total, 4 ACH outside air

Pressure Relationships

- Specified relationship relative to all adjacent spaces
- Operating Rooms: Positive pressure differential of at least +0.01 inWC

Design Space Temperature Requirements

- Specified temperature range for normal operation
- Operating Rooms: 68-75 °F

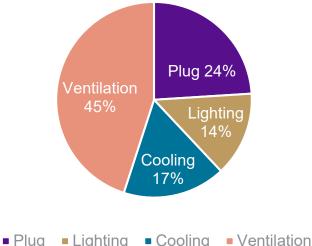
Design Relative Humidity Requirements

- Specified humidity range for normal operation
- Operating Rooms: 20-60 %RH (FGI Guidelines)



Importance of Control Optimization

Average Lab Energy Breakdown



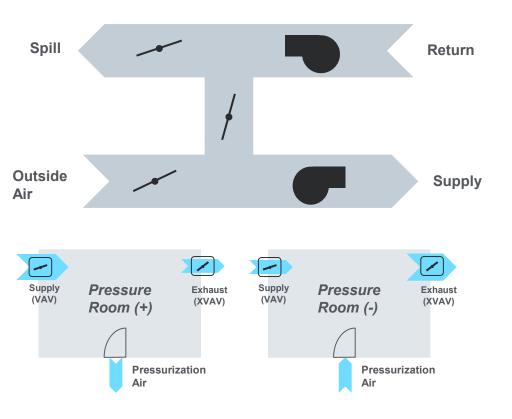
- Critical spaces use a tremendous amount of energy for conditioning
- Many are over ventilated above their require ACH levels to maintain pressure, temperature or humidity
- Temperature and ACH setbacks have potential to provide significant energy savings





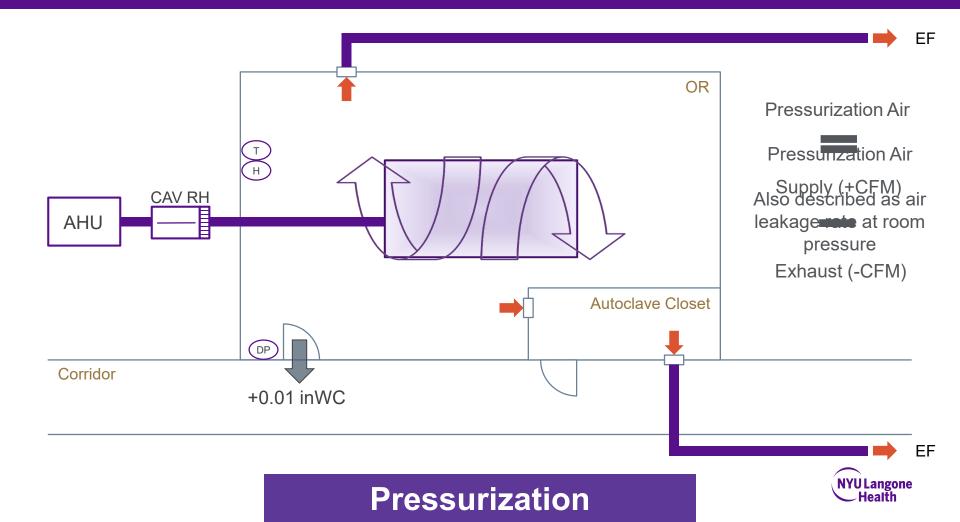
Breakdown of Average Energy Use for Measured Lab Buildings in the USA. Labs 21 Benchmarking Tool, 2010

Key Concepts

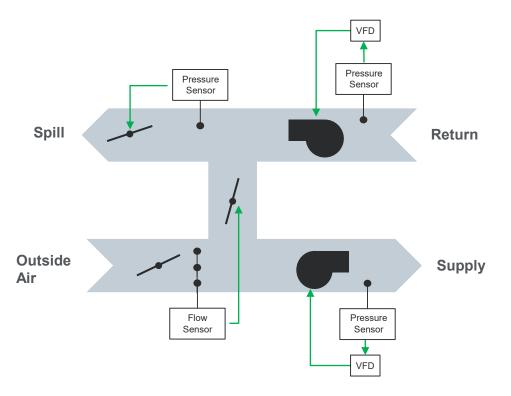


- Pressurization Air = Air In- Air Out = Supply - Return = OA – Spill
- Maintaining pressure often requires excess ACH due to leakage sources:
 - Outlets, Ceilings, Med Gas Manifolds, Access
 Panels
 - <u>Sensors/Tstats</u> Can suck in air resulting in improper control
 - Seal EVERYTHING
- Reduce pressurization air = reduced OA conditioning & fan energy





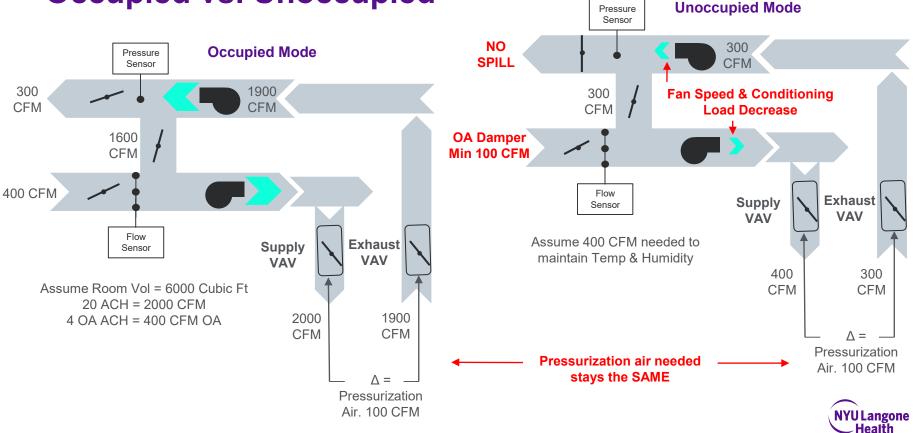
Key Concepts



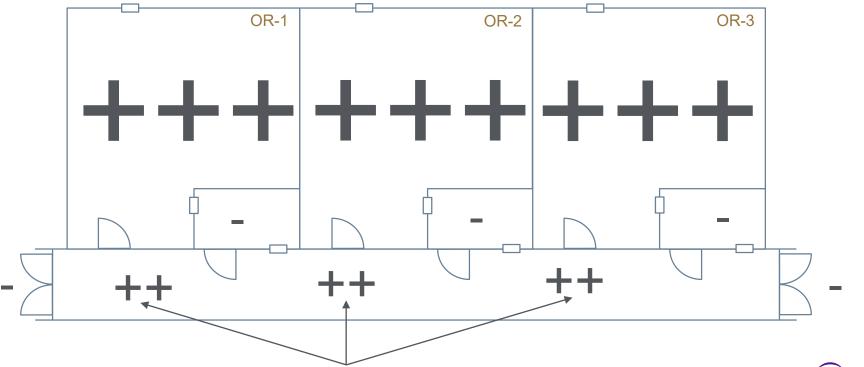
- Outdoor Air Flow Rate = Greater of OA ventilation required for code and make up air (exhaust) or Pressurization Air required
- The goal is to reduce pressurization air as much as possible to maximize return air available and reduce excess OA ventilation. This allows savings in occupied mode and expands savings in unoccupied mode.
- Flow sensor controls return damper
- Pressure sensor controls spill damper
- Spill Air only required if Pressurization Air < OA ventilation code requirement
- VFDs control room pressure & temp independently



Occupied vs. Unoccupied



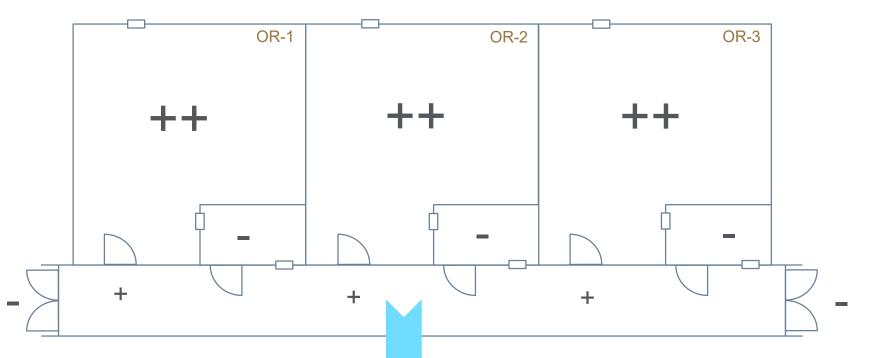
Suite Relationship



Inadequate Corridor Exhaust -ORs "Fight" Each Other to Remain Pressurized



Suite Relationship



Increased Corridor Exhaust = Less Pressurization Air Needed in ORs = Less air to condition



Tisch Hospital – OR Ventilation Project

• Background:

- Original system designed for 48 F constant volume supply 24/7
- Rooms were not tight and air distribution issues, resulting in excessive ACH rates to maintain pressure
- Existing OR suite served by 100% outside air HVAC systems in variety of combinations, including dedicated or shared supply/exhaust, and no individual exhaust control
- Airside systems operate at occupied settings 24/7 although occupancy generally follows a daytime schedule (except emergency surgery)

Project Goals:

- Improve operating room environment through precise control of air flow and performance monitoring
- Reduce energy consumption and costs
- Improve Infection Prevention & Control (IPC) strategies related to mechanical systems and failure scenarios

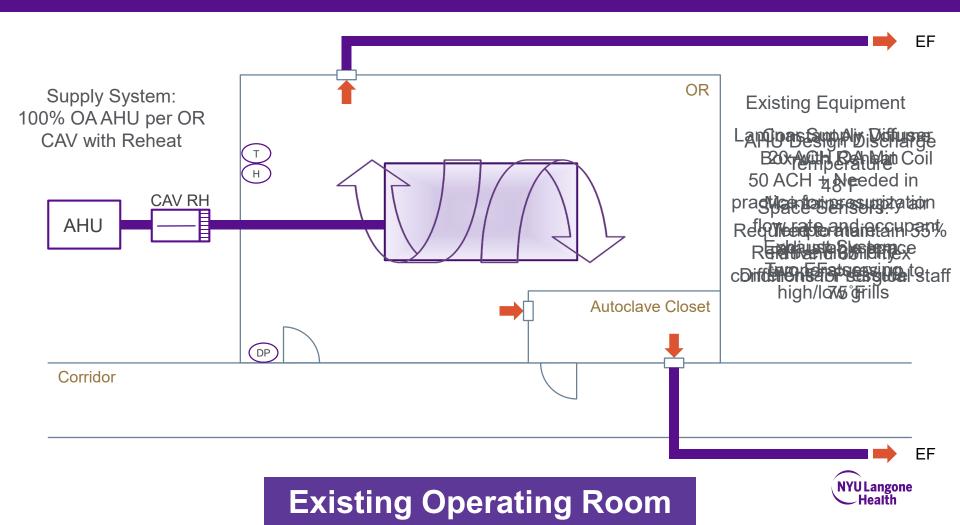


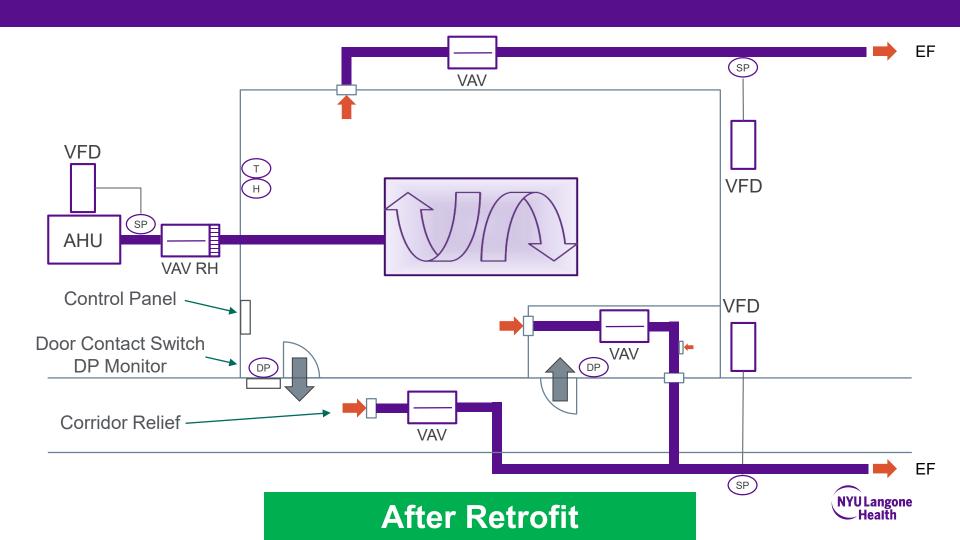
Tisch Hospital – OR Ventilation Project

• Interventions:

- Designed and installed new distributed air volume control systems that can precisely control air flow in and out of individual operating rooms
- Installed corridor exhaust to lower pressure and allow ORs to maintain pressure with fewer ACH
- Installed and integrated new controls, including occupant control panels, temperature/humidity/differential pressure sensors, and door DP monitors for each OR
- Added door contacts to sense and log when operating room doors are open to reduce nuisance pressure alarms, prevent VAV boxes from responding to pressure loss
- Implemented HVAC setback strategies to 6 ACH during unoccupied periods
- Developed new OR Dashboard to monitor compliance conditions
- Introduced the Flex Range to temperature/humidity ranges, which refers to a zone outside of FGI requirements that is deemed acceptable by the IPC board for a temporary period
- Reduce IPC risk present in case of fan failure







Tisch Hospital – OR Ventilation Project

Occupied Operation

Supply and exhaust systems will operate to maintain:

Air flow	20 ACH
Pressure	> 0.02 inWC
Humidity	20-60%
Temperature	Control to SP

CRC Panel will read OCCUPIED

Unoccupied Operation

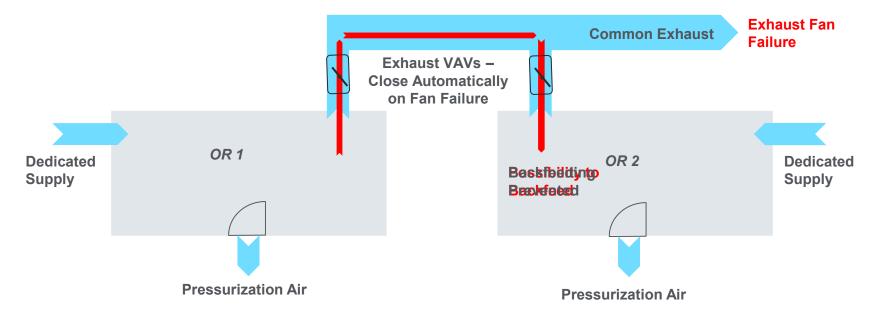
Supply and exhaust systems will operate to maintain:

Air flow Pressure Humidity Temperature Min 6 ACH 0.013 inWC 20-60% Control to 68-75°F Range

CRC Panel will read UNOCCUPIED



Tisch Hospital – Infection Risk Reduction





Automated Daily Performance Review

- Provides comparison of OR space conditions and system performance with NYULH policy and OR Occupancy Program
- Color-coded blocks illustrate hourly performance profile with accompanying tabular summary
- Includes performance metrics to gauge historical performance and encourage continual improvement







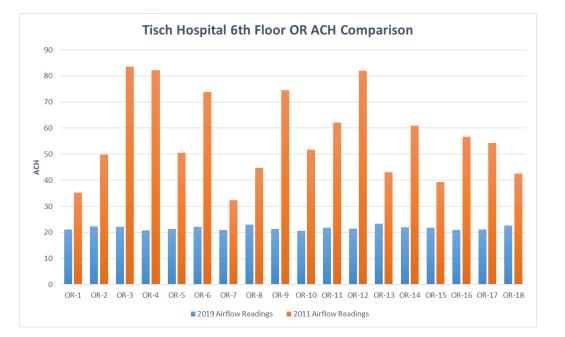


OR #	Trend	6:00 AM	7:00 AM	8:00 AM	9:00 AM	10:00 AM	11:00 AM	12:00 PM	1:00 PM	2:00 PM	3:00 PM	4:00 PM	5:00 PM	6:00 PM	7:00 PM	8:00 PM	9:00 PM	10:00 PM	11:00 PM	12:00 AM	1:00 AM	2:00 AM	3:00 AM	4:00 AM	5:00 AM
OR-1	Haniddy A Personer Temperature	68.6	65.0	L	L	L	L	L	L	65.3	69.0														
OR-2	Hamidily A Pressure Temperature	65.0	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L
OR-3	Hamidily A Personne Temperature	72.0																							
OR-4	Hamidily A Personne Temperature	69.0																							
OR-5	Humidity APressure Temperature	67.0	L	L	L	L	L	L	L	L	L	L	L	L	L	L	65.5	65.0	L	L	L	L	L	L	L
OR-6	Hamidity A Personne Tempredure	69.0		-	68.5	68.0	68.6				-	60.7	68.0			L	L	L	-	-				L	L
OR-7	Hamidily APressure	70.0	67.3	65.3	65.0	\$4,0	- U		L		L	L	L	L			L	L	L	L	L				
OR-8	Traproduce Hamidily A Personer		61.3	65.3	65.0							L.		L.			L	L				L			
OR-9	Trapreslare Hanidity APressure	68.0				L						L		L			L	L							L
OR-10	Traproduce Haniddy APersoner	72.0	66.0	L	— ,	68.3	69.5	72.0																	
OR-11	Traproduce Hamidily APernance	69.0				69.8	70.3	71.0	69.3	69.5	69.0						67.5	67.0	L	L	L	L	L	L	L
OR-12	Traprestare Hamidily APressare	70.0	67.8	67.0				L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L
OR-13	Traprestare Hamidity APressure	68.0		L		L	L	69.0	69.8	68.0	68,5	68.3	72.0	70.3	69.0										
OR-14	Trapreslare Hanidity APressure	71.0											67.5	65.0	L	L	L	L	L	L	L	L	L	L	L
OR-15	Traperature Hanidity APressure	69.0								69.4	70.0														
OR-15	Traprealare Hamidily APressure	65.0	L	L	66.3	68.0		68.5	69.0					67.0	65.0	L	L	L	L	L	L	L	L	L	L
OR-10	Traperature Hamidity	68.3	65.0		65.8	65.0	L	65.3	65.0	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L
OR-17 OR-18	Traproduce Hemidily	66.0	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L	L
	A Pressure Traperature Nanidity	71.7	68.0		68.8	70.4	70.0																		
OR-19	APersone Temperature	72.0	66.0	L	L.	L	L	L	L	L	L	L	L	L	L	L	L.	L	L	L	L	L.	L.	L	L



Tisch Hospital has achieved:

- Drastically reduced occupied ACH rates
- 99.9% humidity compliance
- 98.9% temperature compliance
- 99.6% pressure compliance
- 99.8% AHU runtime
- \$500k in estimated energy savings





Cascaded PID Loops

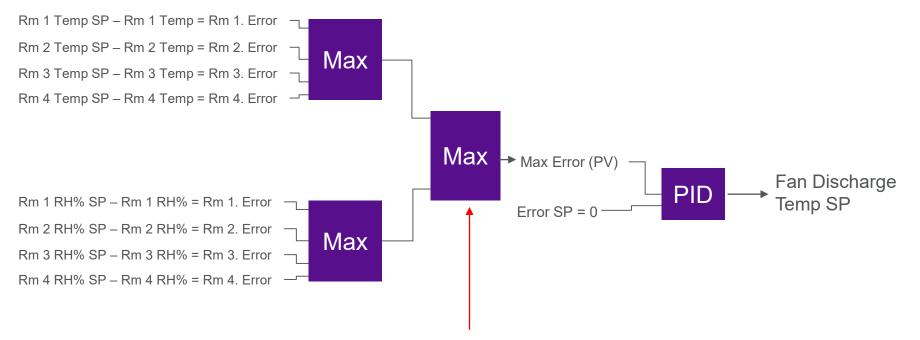
Okay Approach Room Temp PID Reheat Valve Room SP **Better Approach** Room Temp PID **Discharge Temp SP** Room SP Disch. Temp SF PID Reheat Valve Disch. Temp

Other Applications

- Room temp vs RmTmpSp to AHU discharge temp SP for room temperature control
- Room %RH vs RmDeHumSP to AHU discharge temp SP for room dehumidification control
- Room %RH vs RmHumSp to AHU discharge HumSp (or Dewpoint) for room humidity control.
- Differential pressure vs DpSp to GPM setpoint for pump speed control. For parallel pumping of multiple plants.
- Return CO2 vs RCO2Sp to AHU discharge CO2.
- Room temp vs RmTmpSP to VAV box CFMSp



Worst Case Error Control



Temp and RH error are close enough for tuning purposes

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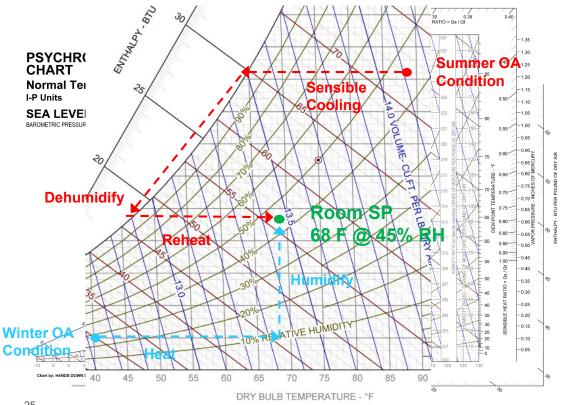
Example - VAV Control Optimization



- Open lab spaces often are designed with multiple Tstats controlling boxes in different zones
- Mixing of air in the space results in divergence. One set of boxes will go to full cooling, one will go to full heating and will swing wildly
- Mitigate by averaging Tstat readouts and controlling to average or controlling to single Tstat.



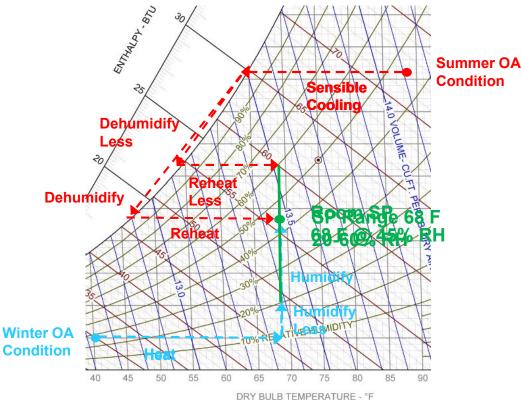
Humidity and Temperature Control – Base Case



- Typical controls control to a temp and humidity set point
- Often requires both latent and sensible heat changes



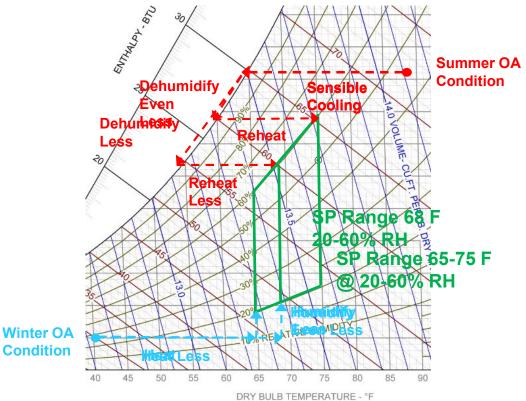
Humidity and Temperature Control – Variable RH



- Occupied Mode: Temperature is set, but humidity operates on a range.
- Reduces latent cooling and reheating and humidification for various temperature and humidity ranges.



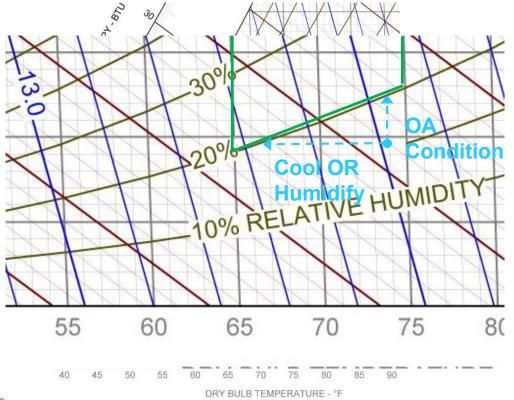
Humidity and Temperature Control - Unoccupied



- In unoccupied mode, temperature becomes a range as well
- Control to zone instead of point
- Allow for elimination of unnecessary conditioning under many circumstances



Humidity and Temperature Control – Fuel Choice



- Certain temperature ranges allow for a choice between sensible cooling or humidification
- Can optimize based on fuel costs
- In our case, always maximizing sensible cooling (i.e. cooling to minimum flex temp) and minimizing humidification results in lowest cost



Next Step – Statistical Management of VAVs

- VAVs modulate and enter reheat depending on AHU discharge temp
- If all VAVs are reheating, discharge temp is too low = Energy Waste
- What if some or no VAVs are reheating? Lowering discharge temp will cause modulating VAVs to close.
 - More reheat on some units, higher CHW consumption
 - AHU fans will slow down, saving fan energy
 - Raising discharge temp will reduce reheating but boxes open, increasing fan energy
- Key is to find balance for optimum cost reduction
- Challenging because controls have to know real time marginal cost of steam and CHW.
- Single starved VAV on a system can force many VAVs to reheat. Add supplemental cooling.







Thank You